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PROBLEM 3.

In class we have discussed the element stiffnesses for both a spring and a rod element, which are given as:

Spring element stiffness matrix:

$$k \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} d_1 \\ d_2 \end{Bmatrix} = \begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix}$$

Rod element stiffness matrix:

$$\frac{AE}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} d_1 \\ d_2 \end{Bmatrix} = \begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix}$$

We will now consider the torsional element (also referred to as a torsional spring) shown in Figure 4, which has the element spring constant given below

$$\frac{JG}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} T_1 \\ T_2 \end{Bmatrix}$$

where J is the polar moment of inertia, G is the shear modulus, and L is the element length. Note that the torsional spring stiffness relates *nodal angle of twist* θ to the *nodal applied torque* T .

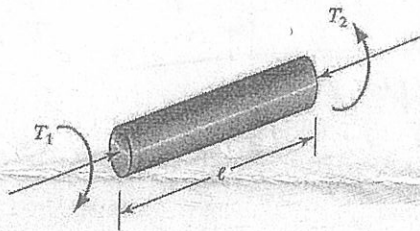


Figure 4. Torsional spring element.

Hint: recall that for a solid cylindrical rod the polar moment of inertia is given as $J = \frac{1}{2} \pi r^4$ where r is the radius of the rod. *Be careful with units!*

For the problem shown in For element 1: $r_1 = 2$ inches, $L_1 = 2$ ft, $G_1 = 3 * 10^6$ lb/in²

For element 2: $r_2 = 1$ inch, $L_2 = 1$ ft, $G_2 = 3 * 10^6$ lb/in²

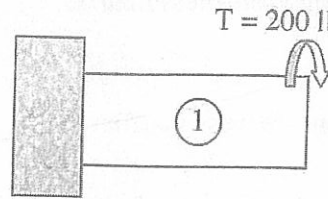
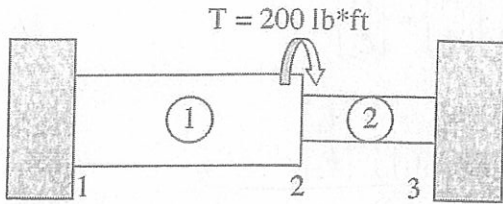
Figure 5a (clearly show and label all work):

1. assemble the *global* stiffness matrix.
2. solve for the angle of twist at node 2.
3. justify your analysis by considering the angle of twist for a torque of 200 lb*ft applied on only element 1. (i.e. see Figure 5b)

Part 3 → Comparison

$$\theta = \frac{TL}{JG} = \frac{(200 \text{ lb}\cdot\text{ft})(12 \text{ in})}{(8\pi \text{ in}^4)(3 \times 10^6 \text{ lb/in}^2)} = 7.64 \times 10^{-4} \text{ rad}$$

§ = $\theta = 7.64 \times 10^{-4} \text{ rad}$
 ↓
 this is more because less resistant to twist in this case!



For element 1: $r_1 = 2$ inches, $L_1 = 2$ ft, $G_1 = 3 \times 10^6 \text{ lb/in}^2$
 For element 2: $r_2 = 1$ inch, $L_2 = 1$ ft, $G_2 = 3 \times 10^6 \text{ lb/in}^2$

Figure 5a. (left) Torsional problem via the finite element method. (right) Comparison problem.

El 1: $\frac{JG}{L} = \frac{(8\pi \text{ in}^4)(3 \times 10^6 \text{ lb/in}^2)}{2 \text{ ft} \times \frac{12 \text{ in}}{1 \text{ ft}}} = \pi \times 10^6 \text{ lb}\cdot\text{in} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$

$J_1 = \frac{1}{2} \pi (2)^4 = 8\pi \text{ in}^4$

El 2: $\frac{JG}{L} = \frac{(\frac{\pi}{2} \text{ in}^4)(3 \times 10^6 \text{ lb/in}^2)}{1 \text{ ft} \times \frac{12 \text{ in}}{1 \text{ ft}}} = \frac{\pi}{8} \times 10^6 \text{ lb}\cdot\text{in} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$

$J_2 = \frac{1}{2} \pi (1)^4 = \frac{\pi}{2} \text{ in}^4$

GLOBAL STIFFNESS MATRIX

$$\begin{bmatrix} T_1 \\ T_2 \\ T_3 \end{bmatrix} = \pi \times 10^6 \text{ lb}\cdot\text{in} \begin{bmatrix} 1 & -1 & 0 \\ -1 & 1 + \frac{1}{8} & -\frac{1}{8} \\ 0 & -\frac{1}{8} & \frac{1}{8} \end{bmatrix} \begin{bmatrix} \theta_1 \\ \theta_2 \\ \theta_3 \end{bmatrix}$$

§ $T_2 = \frac{9}{8} \pi \times 10^6 \theta_2$

$200 \text{ lb}\cdot\text{ft} \times \frac{12 \text{ in}}{1 \text{ ft}} = (\frac{9}{8} \pi \times 10^6) (\theta_2) \Rightarrow \theta_2 = 6.79 \times 10^{-4} \text{ rad}$

Angle of twist at node 2