

# Internal Pipeflow Problem Using a Finite Element Approach

Professor Frank Fisher  
Department of Mechanical Engineering  
Stevens Institute of Technology

## ME345: Modeling and Simulation

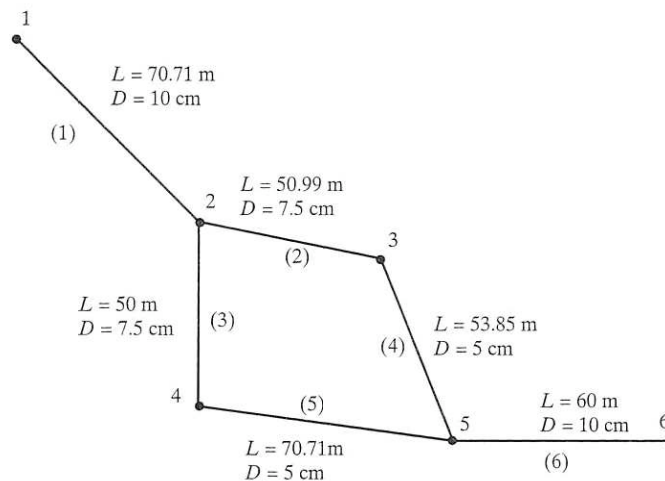
For internal pipe flow, one can derive a finite element formulation that relates the *volumetric flow rate at a point  $Q$*  [units of  $m^3/s$ ] to the *nodal pressures  $p$*  [units of  $Pa$ ] via an element stiffness matrix of the form

$$\frac{\pi D^4}{128 L \mu} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} p_1 \\ p_2 \end{Bmatrix} = \begin{Bmatrix} Q_1 \\ Q_2 \end{Bmatrix}$$

where  $D$  is the internal diameter of the pipe,  $L$  is the length of the element, and  $\mu$  is the dynamic viscosity of the fluid [units of  $s/m^2$ ]. (The above relationship holds true for simple, laminar flows.) The coefficient  $(\pi D^4)/(128 L \mu)$  is sometimes known as the *flow resistance* [units of  $m^5 / N \cdot s$ ].

Using this definition of 1D pipe flow, analyze the following piping schematic below given the following known (boundary) conditions:

$$Q_1 = Q_{\text{inlet}} = 5 \times 10^{-4} \text{ m}^3/\text{s}$$
$$\mu = 0.3 \text{ N s} / \text{m}^2$$
$$p_6 = 0 \text{ Pa}^1$$



<sup>1</sup> Here we are treating  $p_6$  as at atmospheric pressure, and can assign it an arbitrary value; setting it equal to zero is the easiest value to set it to here. All other pressures determined will be relative to this reference pressure.