

Internal Pipeflow Problem Using a Finite Element Approach (Example 2)

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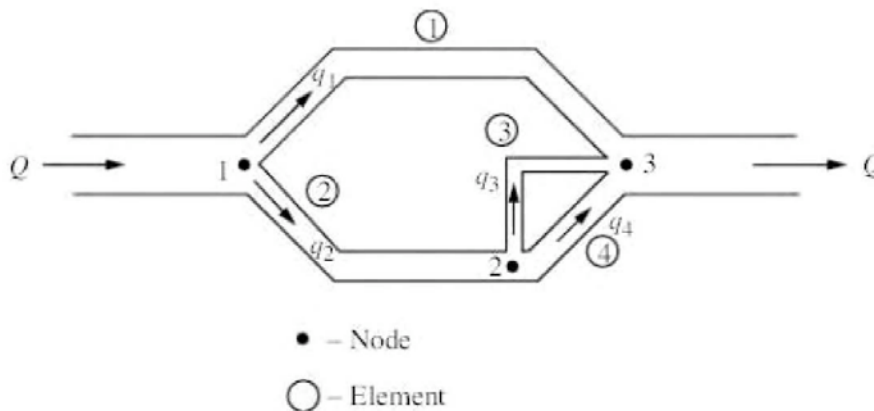
For internal pipe flow, one can derive a finite element formulation that relates the *volumetric flow rate at a point Q* [units of m^3/s] to the *nodal pressures p* [units of Pa] via an element stiffness matrix of the form

$$\frac{\pi D^4}{128 L \mu} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} p_1 \\ p_2 \end{Bmatrix} = \begin{Bmatrix} Q_1 \\ Q_2 \end{Bmatrix}$$

where D is the internal diameter of the pipe, L is the length of the element, and μ is the dynamic viscosity of the fluid [units of s/m^2]. (The above relationship holds true for simple, laminar flows.) The coefficient $(\pi D^4)/(128 L \mu)$ is sometimes known as the *flow resistance* [units of $m^5 / N s$].

Using this definition of 1D pipe flow, analyze the following piping schematic below given the following known (boundary) conditions:

$$\begin{aligned} Q_1 &= 0.1 \text{ m}^3/\text{s} \\ \mu &= 0.96 * 10^{-3} \text{ N s} / \text{m}^2 \\ p_3 &= 0 \text{ Pa}^1 \end{aligned}$$



¹ Here we are treating p_3 as at atmospheric pressure, and can assign it an arbitrary value; setting it equal to zero is the easiest value to set it to here. All other pressures determined will be relative to this reference pressure.